AG-Structural Integrity + UAG Fatigue, 4. April 2019, Steyr, 25 min + 5 min



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Novel Fatigue Lifetime Prediction for Brittle Materials

by using Strength Failure Mode-linked Modeling of Loading, a Basic S-N Curve,

and Application of a Strength Mode-linked variation of Kawai's 'Modified Fatigue Strength Ratio' for estimating further S-N curves

- brittle material behavior such as UD (here), grey cast iron, and concrete -

1 Introduction to Static and Fatigue Design

- 2 Cuntze's *Failure Mode Concept-based* Strength Criteria
- 3 Cuntze's Fatigue Lifetime Prediction Concept
- 4 Generation and Novel Interpretation of UD Haigh Diagrams
- **5** Steps of the Proposed *Fatigue Lifetime Prediction Concept*

Presentation of never funded hobby-investigations.

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Flow Diagram: Structural Design and Design Verification Nachweise



How may one principally discriminate *Material Behaviour*?



STATIC :

Reserve Factor is load-defined :

Material Reserve Factor :

$$RF = \frac{\text{Predicted Failure Load}}{j \cdot \text{Design Limit Load}} > 1.$$

 $f_{\text{Res}} = \frac{\text{Strength Design Allowable}}{\text{Stress at } j \cdot \text{Design Limit Load}} > 1.$

CYCLIC :

- *RF_{life, Predicted Lifetime}* Determination of **Inspection time**
- Determination of Replacement time

$$RF_{life} \approx \frac{\text{Predicted Lifetime}}{j_{life} \cdot \text{Design Limit Lifetime}} > 1.$$

Global versus Modal Strength Failure Conditions SFCs (criteria)



Observed ! Fracture Morphology of Transversely-isotropic UD Material



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Basic Features of Cuntze's Failure-Mode-Concept (FMC), 1995

<u>plus</u> the fact, that transversely-isotropic UD materials exhibit a '5-fold' material symmetry characteristic = 5 Strengths, 5 Failure Modes, 5 E, etc.

• Each failure mode represents 1 independent failure mechanism

and thereby 1 piece of the complete failure surface

- Each failure mechanism is governed by 1 basic strength (is observed!)
- Each failure mode can be represented by 1 failure condition.

Therefore, equivalent stresses can be computed for each mode !! This is of advantage when deriving S-N curves and Haigh diagrams with minimum test effort. Understanding the terms Material Stressing Effort and Equivalent Stress

Helpful "To turn the right screw" in design is the delivery of

equivalent stresses and of material stressing efforts Eff

mode material stressing effort * (in German "Werkstoffanstrengung")



• material stressing effort *Eff* = *artificial technical term*, *created together with QinetiQ, UK, during the World-Wide-Failure-Exercises*

Cuntze's Set of Modal 3D UD Strength Failure Conditions ('criteria')

Cuntze = 'simple Mises ' amongst the UD criteria

Invariants, replaced by their stress formulations !!

FF1
$$Eff^{\parallel\sigma} = \overline{\sigma}_{1}/\overline{R}_{\parallel}^{t} = \sigma_{eq}^{\parallel\sigma}/\overline{R}_{\parallel}^{t}$$
, $\overline{\sigma}_{1} \cong \varepsilon_{1}^{t} \cdot E_{\parallel} *$ [Cun04,
 $\overline{\sigma}_{1} \cong \varepsilon_{1}^{t} \cdot E_{\parallel} *$ Cun11]
FF2 $Eff^{\parallel r} = -\overline{\sigma}_{1}/\overline{R}_{\parallel}^{c} = +\sigma_{eq}^{\parallel r}/\overline{R}_{\parallel}^{c}$, $\overline{\sigma}_{1} \cong \varepsilon_{1}^{c} \cdot E_{\parallel}$ $2 \frac{filament}{modes}$
IFF1 $Eff^{\perp \sigma} = [(\sigma_{2} + \sigma_{3}) + \sqrt{(\sigma_{2} - \sigma_{3})^{2} + 4\tau_{23}^{2}}]/2\overline{R}_{\perp}^{t} = [\sigma_{eq}^{\perp\sigma}/\overline{R}_{\perp}^{t}]$ 3 matrix
IFF2 $Eff^{\perp r} = [(\frac{\mu_{\perp \perp}}{1 - \mu_{\perp \perp}}) \cdot (\sigma_{2} + \sigma_{3}) + \frac{1}{1 - \mu_{\perp \perp}} \sqrt{(\sigma_{2} - \sigma_{3})^{2} + 4\tau_{23}^{2}}]/\overline{R}_{\perp}^{c} = +\sigma_{eq}^{\perp r}/\overline{R}_{\perp}^{c} modes$
IFF3 $Eff^{\perp \parallel} = \{[\mu_{\perp \parallel} \cdot I_{23-5} + (\sqrt{\mu_{\perp \parallel}^{2}} \cdot I_{23-5}^{2} + 4 \cdot \overline{R}_{\perp \parallel}^{2} \cdot (\tau_{31}^{2} + \tau_{21}^{2})^{2}]/(2 \cdot \overline{R}_{\perp \parallel}^{3})\}^{0.5} = \sigma_{eq}^{\perp \parallel}/\overline{R}_{\perp \parallel}$
with $I_{23-5} = 2\sigma_{2} \cdot \tau_{21}^{2} + 2\sigma_{3} \cdot \tau_{31}^{2} + 4\tau_{23}\tau_{31}\tau_{21}$

Interaction of modes:

$$Eff^{m} = (Eff^{||\tau})^{m} + (Eff^{||\sigma})^{m} + (Eff^{\perp\sigma})^{m} + (Eff^{\perp\tau})^{m} + (Eff^{\perp||})^{m} = 1$$

with mode-interaction exponent

2.5 < m < 3 from mapping tests data

Typical friction value data range: see [Pet16] for measurement

 $0.05 < \mu_{\perp \parallel} < 0.3, \quad 0.05 < \mu_{\perp \perp} < 0.2$

Poisson effect * : bi-axial compression strains the filament without any σ_1 t:= tensile, c: = compression, || : = parallel to fibre, \perp := transversal to fibre

 τ_{23}

Хз 🖡

Modal treatment requires an Interaction of the Single SFCs

In the FMC:I

Interaction of adjacent Failure Modes in the mode transition zones

= by a 'series failure system' model that considers an

'Accumulation' of interacting mode- associated failure danger portions $\mathit{Eff}^{\mathrm{mode}}$

$$Eff = \sqrt[m]{(Eff^{\text{mode 1}})^m + (Eff^{\text{mode 2}})^m +} = 1 = 100\%$$
, if failure

with mode-interaction exponent *m*, from mapping experience

It is assumed engineering-like: m takes the same value for all mode transition zones captured by the interaction formula above

Visualization of Interaction: example UD Failure Modes $\breve{\sigma}_1 = 0$

 $\tau_{21}(\sigma_2)$ or $\{\sigma\} = (0, \sigma_2, 0, 0, 0, \tau_{21})^T$



IFF 1-2-3 Cross-section of the Fracture Failure Surface (body)



* Above tested were so-called isolated test specimens.
 * For the presented fatigue approach embedded laminas are to consider!

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 $2D \implies 3D$ Fracture surface by replacing the stress by the equival. stress



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"Fatigue is the black art,

to produce financial black holes"

[J. Draper]

Therefore, in order to reduce very costly cyclic laminate test programs

the German Academic Research Group BeNa (proof of service strength),

founded by the author in 2010,

aims at :

A failure mode-based Lifetime Prediction Method, lamina-oriented on the embedded lamina in order to capture in-situ effects and using <u>failure mode-based S-N curves</u>.

Stress Life Fatigue ← approaches → Strain Life fatigue (*ductile behaviour*)

Fatigue limit, endurance limit, and fatigue strength: expressions, used to describe a cyclic property of a material

Behaviour:

• brittle : max stress (Oberspannung), σ_{max} is responsible for damaging

•ductile: amplitude stress σ_a is responsible for damaging (slip)

S-N curve (Wöhlerkurve): $R = \sigma_{min} / \sigma_{max}$

$\sigma_a \,$ and σ_{max} (if brittle) are used

stress-life fatigue curve of a material, in terms of fracture cycles *N*, for a distinct applied stress $S \equiv \sigma(N)$. (Note: Renders the weakening of a repeatedly loaded material)

Haigh Diagram:

stress amplitude $\sigma_a(\sigma_m, R)$ is used

Mean stress σ_m influence f_M of isotr. materials: prediction on basis of 2 test points ($\sigma_a_{R=-1}$, 0), $\sigma_a_{R=0}$, $\sigma_a_{R=0}$), $f_M = \sigma_a_{R=-1} / \sigma_a_{R=0}$, represents the slope

State of the Art : <u>Cyclic</u> Strength Analysis of UD-ply composed Laminates

• No Lifetime Prediction Method available, applicable to any Laminate

- Procedures base as with metals on stress amplitudes and mean stress correction f_M
- Procedures base on specific laminates and therefore cannot be generally applied
- <u>Up to now</u>: Engineering Approach <u>Static Design Limit Strain</u> of $\varepsilon < 0.3\%$, practically means negligible matrix-microcracking. Design experience proved: <u>No</u> fatigue danger given
- <u>Future</u>: Design Limit Strain shall be increased (EU-project: MAAXIMUS)
 Beyond ε ≈ 0.5% first filament breaks, diffuse matrix-microcracking changes to a discrete localized one.

Often, fiber-dominated laminates are used in high performance stress applications. UD-material should be better exploited !

Fatigue Damaging Drivers of Ductile and Brittle behaving Materials

• Ductile Material Behavior (example: isotropic metal materials)

1 damaging mechanism *acts* = "*slip band shear yielding*"

drives damaging under cyclic tensile, compressive, shear and torsional stresses:

Therefore, this single mechanism can be described by one single strength formulation:

the Mises Yield failure condition!

• Brittle Behaving Material Behavior : isotropic Materials

2 damaging driving mechanisms act = Normal Fracture failure mode (NF), Shear Fracture failure (SF)

• Brittle Behaving UD Material Behavior : transversely-isotropic UD Materials

5 damaging driving fracture failure mechanisms act \equiv 5 Fracture failure modes

Assumption:

"If the failure mechanism (mode) cyclically remains the same as in the static case t*hen*

- the fatigue damaging driving failure parameters are the same and
- the applicability of static SFCs is allowed for quantifying damaging portions !"

Cyclic development of damaging, average <u>S-N-curve</u>, brittle material



S-N-mapping of brittle materials: use of σ_{max} is advantageous compared to the amplitude σ_a !

Lin-Log S-N Curve: Average Curve (mapping) and Design Curve (verification)



FF:= fibre failure. IFF:= Inter Fibre Failure, CDS:= characteristic damage state at the end of diffuse damaging

1 Input

Operational Loading: Load time curves (modeling rain flow, ..)

Time domain: Cycle-by-cycle or block-by-block (less computation effort)

Frequency dom.: Load spectra (loss of Reihenfolge)

Safety Concept: Design safety factor Life $j_{Life} = 3 - 10$, or

an Inspection interval, or an Replacement time approach

- 2 Transfer of operational loadings into stresses by using structural analysis
- 3 Domains of Fatigue Analysis

LCF: high stressing,
HCF: intermediate stressing 10.000 < n < 1.000.000, rotor tube
VHCF: low stressing and strains (SPP1466) > 10.000.000 centrifuges, wind rotor blades

4 Provision of Haigh Diagrams which involve all necessary S-N curves with generation of 'Constant Fatigue Life (CFL) curves'

3 Haigh Diagrams are provided : FF1-FF2, IFF1-IFF2, IFF3

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How look Kawai model-predicted 'Mode S-N curves ?



Kawai's Modified Fatigue Strength Ratio Ψ mode-linked applied to predict further necessary S-N curves on basis of one measured Master S-N curve R = 0.1 (NF), 10

(SF) Searched model Table : Formulas to map the basic S-N curve and Kawai's Master Ψ -model * Mapping function for the basic S-N curve: $\sigma_{\max}(N) = c1 + \frac{\overline{R}_m - c1}{\left(\frac{\log N}{c^3}\right)^{c^2}}$, $\sigma_{\min}(N) = c1 + \frac{-\overline{R}_m - c1}{\left(\frac{\log N}{c^3}\right)^{c^2}}$ * Relationships: $R = \sigma_{min}/\sigma_{max} = (\sigma_m - \sigma_a)/(\sigma_m + \sigma_a)$ $\sigma_{01}(N) = basic \sigma_{max}(N)$ $\sigma > 0$: $\sigma_{a} = 0.5 \cdot \sigma_{max} \cdot (1 - R)$, $\sigma_{m} = 0.5 \cdot \sigma_{max} \cdot (1 + R) = \sigma_{max} - \sigma_{a}$ $\sigma_{10}(N) = basic \sigma_{min}(N)$ $\sigma < 0$: $\sigma_a = -0.5 \cdot \sigma_{min} \cdot (1 - 1/R)$, $\sigma_m = 0.5 \cdot \sigma_{max} \cdot (1 + R) = \sigma_{min} + \sigma_a$ $\sigma_{\max}(N, R) = \Delta \sigma / (1-R) \equiv [2 \cdot \sigma_a / (1-R)]$ with $\Delta \sigma = \text{stress range}$, strength value $R_m = \sigma_{\max} (n = N = 1)$ * Definition of Kawai's 'modified fatigue strength ratio' (valid for each failure domain, after Cuntze) **FF1**: $\sigma > 0$: $\Psi t = \sigma_a / (R_{||}^t - \sigma_m) = 0.5 \cdot (1 - R) \cdot \sigma_{max} / [R_{||}^t - 0.5 \cdot (1 + R) \cdot \sigma_{max}]$ or FF1 FF2 = $0.5 \cdot (1-R) \cdot Eff^{|\sigma|} [1 - 0.5 \cdot (1+R) \cdot Eff^{|\sigma|}$ with $\sigma_{max} > \sigma_{min}$ **FF2:** $\sigma < 0$: $\Psi_c = \sigma_a / (R_{\parallel}^c + \sigma_m) = 0.5 \cdot (1-R) \cdot \sigma_{min} / [R_{\parallel}^c - 0.5 \cdot (1+R) \cdot \sigma_{min}]$ with $|\sigma_{min}| > |\sigma_{max}|$ 'master * Derivation of Kawai's 'master modified fatigue strength ratio' using 'basic mode S-N curve' $\underline{\sigma > 0}: \quad \Psi t \text{ master}(n) = 0.5 \cdot (1 - Ro1) \cdot \sigma_{o1}(N) / [R_{II}^{t} - 0.5 \cdot (1 + Ro1) \cdot \sigma_{o1}(N) \text{ with } \sigma_{max} = \sigma_{o1}, Ro1 = 0.1$ FF1 FF2 $\sigma > 0$: $\Psi c master(n) = (1-R_{10})/[1+R_{10}+2R_{II} \cdot R_{10}/\sigma_{10}(N)]$ with $\sigma_{min} = \sigma_{10}$, $R_{10}=10$ * Derivation of other relevant S-N curves in the two modes FF1 and FF2 FF1 $\sigma_{max}(R,N) = (2 \cdot R_{II}^{t} \cdot \Psi_{t master}) / [\Psi_{t master} - R + R \cdot \Psi_{t master} + 1],$ FF2: $\sigma_{min}(R,N) = -(2 \cdot R_{\parallel}^{\circ} \cdot \Psi c_{master}) / [\Psi c_{master} + R + R \cdot \Psi c_{master} - 1]$ 26

S-N curve modelled : linearly, non-linearly in semi-log, log-log diagrams

Example: Individually lin-log mapped <u>FF1-FF2</u>-linked S-N curves



Rigorous Interpretation of the Haigh diagram: example FF1-FF2 UD

displaying failure mode domains and transition zone, test data [Hah14]



NF = Normal Fracture, SF = Shear Fracture, N = fracture cycle number, CFL = Constant Fatigue Life

IFF3 UD Haigh diagram,

Display of a two-fold mode effect (a:= amplitude, m:= mean, N := number of fracture cycles, R := strength and R := $\sigma_{min}/\sigma_{max}$). Test data CF/EP, courtesy [Hahne14]



Mapped lin-log IFF1-IFF2-linked S-N curves [data, courtesy C. Hahne]



How to obtain <u>CFL</u> curves in the Transition Domain ?

* There is no problem to establish the Haigh diagrams FF and IFF3 due to the strength values being of similar size in each case: The static interaction formula was sufficient.
 * For a Haigh Diagram for really brittle materials, when Rtrans is very different to -1, a new solution procedure had to be used. Chosen was an exponentially decaying function. that practiscally ends where the other mode reigns



 \rightarrow failure domain-linked <u>constant fatigue life (CFL</u>) curves: $\sigma_a(\sigma_m, R, N=constant)$

IFF1- IFF2 UD Haigh diagram

displaying the failure mode domains, transition zone



- Curve in the IFF1 domain looks non-linear !
- Check points from Ψ -prediction lie higher than points from S-N test data evaluation

(The computed S-N curve X-points are anchor (checking) points for the to be predicted CFL curves)

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Novel ? idea: Failure mode-wise modelling of Loading Cycles for high-performance 'fiber-dominated designed', UD laminas-composed laminates

For simply displaying the approach *it is chosen* : a *loading* R = -1

Separation due to the activated inherent different failure modes



Step 1 : Failure mode-linked apportionment of cyclic loading (novel)

A specific rain-fall procedure must be applied

Example: FF1 failure mode





Step 3: Application of Kawai's 'Modified Fatigue Strength Ratio'.

* improvable in the intermediate R-range

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 $D (FF1, FF2) = NF : (n_1 / N_1 + n_2 / N_2 + n_3 / N_3) + SF : (n_4 / N_4)$ $\Rightarrow D = D (FF1, FF2) + D (IFF1, IFF2, IFF3) \leq D_{feasible}$ from test experience **Step 4: Determination of Damaging Portions by Static Strength Criteria**

Step 5: <u>Mode-wise</u> Accumulation of Damaging Portions (<u>novel</u>)

What was the main Objective of this Investigation ? on basis of the 'rigorous failure -mode thinking'

For high performance composite parts:

Fatigue pre-dimensioning procedure for

'well-designed', UD laminas-composed laminates just by

- single lamina-dedicated mode-representative Master S-N curves, derived from *sub-laminate* test specimens, which capture the embedded ply (in-situ) effects,
 - * further necessary Kawai-model-predicted S-N curves

* automatically derivable CFL curves of the Haigh diagrams.

Everything in the world is terminated by **chance** and **fatigue**. Heinrich Heine

Thank You !

Literatur download possible from carbon~connected.de/Group/CCeV.Fachinformationen/Mitglieder

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- Material : homogenized (smeared) model of the envisaged complex material which might be a material combination
- Failure : structural part does not fulfil its functional requirements such as FF = fiber failure, IFF = inter-fiber-failure (matrix failure), leakage, deformation limit (tube widening, delamination size limit, ..) ⇒ = a project-defined 'defect'
- Fatigue : process, that degrades material properties
- Fatigue Life Stages (1) accumulation of damaging until initiation of a critical damage size (classical fatigue life prediction domain), (2) damage growth until onset of final fracture (domain of damage tolerance concepts), (3) separation (not of interest)
- **Damaging** (not also damage, as used in English literature) : process wherein the results, the damaging portions, finally accumulate to a damage size such as a macro-scopic delamination. Accumulation tool usually used is *Palmgren-Miner's Damaging* Accumulation Rule (= model)
- **Damage** : sum of the accumulated damaging or an impact failure, that is judged to be critical. Then, *Damage Tolerance Analysis* is used to predict damage growth under further cyclic loading or static failure under Design Ultimate Load

Haigh Diagram : involves all S-N curves required for fatigue life prediction.40

Static strength failure $Eff = [(Eff^{NF})^m + (Eff^{SF})^m]^{m^{-1}} = 100\%$

$$\left(\frac{-(\sigma_{2m}-\sigma_{2a})+|\sigma_{2m}-\sigma_{2a}|}{2\cdot\overline{R}_{\perp}^{c}}\right)^{m}+\left(\frac{\sigma_{2m}+\sigma_{2a}+|\sigma_{2m}+\sigma_{2a}|}{2\cdot\overline{R}_{\perp}^{t}}\right)^{m}=1$$

The used static procedure still works for N = 1 with the interaction formula above delivering the CFL curve for N = 1 cycle, activating both NF + SF .

For higher N the interaction formula is engineering-like simplified. *It reads*:

$$\sigma_{a}(\sigma_{m}) = \left[\left(\frac{\sigma_{aSF}}{\frac{c_{1SF} + \sigma_{m}}{c_{2SF}}} \right)^{m} + \left(\frac{\sigma_{aNF}}{\frac{c_{1NF} + \sigma_{m}}{c_{2NF}}} \right)^{m} \right]^{1/m}$$

Thereby, an exponential decay function of the SF mode CFL curve for SF from $R = \infty$ down to zero at the end of the NF CFL curve at R = 0 is applied:

$$f = \left\lfloor \frac{1}{1 + e^{\frac{c_1 + \sigma_m}{c_2}}} \right\rfloor$$